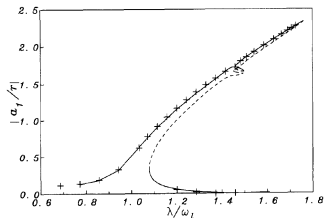
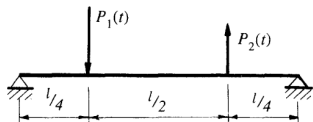


A test example on a homogeneous beam



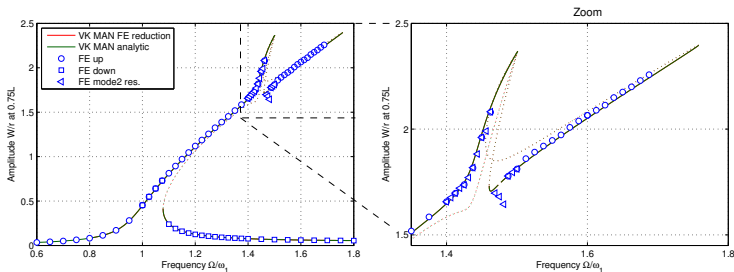
► Details

- A literature example [Lewandowski, 1996]
- Simply-supported beam
- Mass proportional damping ($\xi_k = \alpha / \omega_k$)
- Two opposed harmonic point forces:

$$P_1(t) = 13.63 \frac{EI r}{L^3} \cos \Omega t, \quad P_2(t) = -9.62 \frac{EI r}{L^3} \cos \Omega t,$$

N.L. frequency response around mode 1

Model validation



► Perfect match between 3 models:

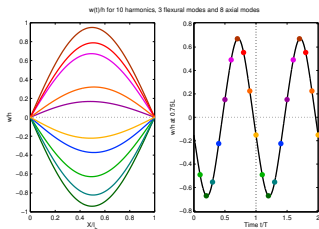
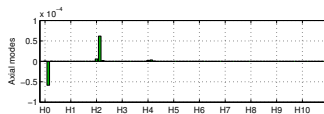
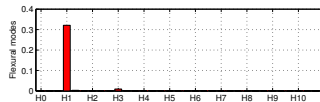
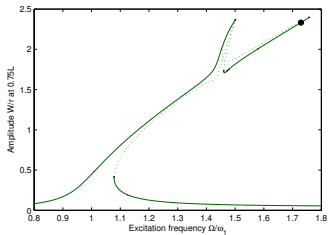
- analytical model + continuation
- full large rotation FE model + time integration
- present FE reduced order model (3 transverse mode, 8 axial modes)

► Special 1/3 superharmonic resonance of mode 2:

$$\omega_2 = 4\omega_1, \quad \Omega \simeq \frac{\omega_2}{3} = 1.33\omega_1$$

N.L. frequency response around mode 1

Oscillations at resonance



N.L. frequency response around mode 1

N.L. coupling with mode 2 ($\Omega \simeq \omega_2/3 = 1.33\omega_1$)

